

Solutions, 316-VI

February 5, 2003

1 Problem 4.6.7

Find the general solution of

$$4y'' - 4y' + 26y = 0 .$$

Solution:

$$\begin{aligned} 26y &= 26e^{rt} \\ + &+ \\ -4y' &= -4re^{rt} \\ + &+ \\ 4y'' &= 4r^2e^{rt} \\ = &= \\ 0 &= (4r^2 - 4r + 26)e^{rt} \end{aligned}$$

Then, we factor

$$4r^2 - 4r + 26 \Rightarrow r_{\pm} = \mu \pm \nu i = 2 \pm 10i$$

so that the general solution is

$$\begin{aligned} y(t) &= Ae^{\mu t} \cos(\nu t) + Be^{\mu t} \sin(\nu t) \\ &= Ae^{2t} \cos(10t) + Be^{2t} \sin(10t) \end{aligned}$$

2 Problem 4.6.21

Solve the IVP

$$y'' - 2y' - 2y = 0, \quad y(0) = 2, \quad y'(0) = 1.$$

Solution:

$$\begin{aligned} 2y &= 2e^{rt} \\ + &+ \\ 2y' &= 2re^{rt} \\ + &+ \\ y'' &= r^2e^{rt} \\ = &= \\ 0 &= (r^2 + 2r + 2)e^{rt} \end{aligned}$$

Then, we factor

$$r^2 + 2r + 2 \Rightarrow r_{\pm} = -1 \pm i$$

so that the general solution is

$$\begin{aligned} y(t) &= Ae^{-t} \cos t + Be^{-t} \sin t \\ y'(t) &= Ae^{-t}(-\cos t - \sin t) + Be^{-t}(-\sin t + \cos t) \end{aligned}$$

Then

$$y(0) = 2 = A, \quad y'(0) = 1 = -A + B \Rightarrow B = 3$$

giving for the solution:

$$y(t) = 2e^{-t} \cos t + 3e^{-t} \sin t$$

3 Problem 4.6.28

To see the effect of changing the parameter b in the problem

$$y'' + by' + 4y = 0; \quad y(0) = 1, \quad y'(0) = 0$$

solve the problem for $b = 5, 4, 2$ and sketch the solutions. **Solution:**

$$\begin{aligned}
 4y &= 4e^{rt} \\
 + &+ \\
 by' &= bre^{rt} \\
 + &+ \\
 y'' &= r^2e^{rt} \\
 = &= \\
 0 &= (r^2 + br + 4)e^{rt}
 \end{aligned}$$

and factoring:

$$r_{\pm} = \frac{-b \pm \sqrt{b^2 - 16}}{2} = \frac{-b \pm \sqrt{(b-4)(b+4)}}{2}$$

so that the equation has two real, distinct roots for $b < -4$ or $b > 4$, two real equal roots (double root) for $b = \pm 4$ and two complex conjugate roots for $-4 < b < 4$. Therefore the solutions for the 3 cases given ($b = 5, 4, 2$) are:

$$\begin{aligned}
 b = 5 &\Rightarrow r_+ = -1, r_- = -4 \\
 y(t) &= Ae^{-t} + Be^{-4t} \quad ; \quad y(0) = 1 \\
 y'(t) &= -Ae^{-t} - 4Be^{-4t} \quad ; \quad y'(0) = 0 \\
 A + B = 1, \quad -A - 4B = 0 &\Rightarrow A = \frac{4}{3}, B = \frac{-1}{3} \\
 y(t) &= \frac{4}{3}e^{-t} - \frac{1}{3}e^{-4t} \quad .
 \end{aligned}$$

$$\begin{aligned}
 b = 4 &\Rightarrow r_{\pm} = -2 \text{ (double root)} \\
 y(t) &= Ae^{-2t} + Bte^{-2t} \quad ; \quad y(0) = 1 \\
 y'(t) &= -2Ae^{-2t} + Be^{-2t}(1 - 2t) \quad ; \quad y'(0) = 0 \\
 A = 1, \quad -2A + B = 0 &\Rightarrow A = 1, B = 2 \\
 y(t) &= e^{-2t} + 2te^{-2t} \quad .
 \end{aligned}$$

$$\begin{aligned}
 b = 2 &\Rightarrow r_{\pm} = -1 \pm i\sqrt{3} \\
 y(t) &= Ae^{-t} \cos(\sqrt{3}t) + Be^{-t} \sin(\sqrt{3}t) \quad ; \quad y(0) = 1, y'(0) = 0
 \end{aligned}$$

$$y'(t) = Ae^{-t} \left(-\cos(\sqrt{3}t) - \sqrt{3} \sin(\sqrt{3}t) \right) + Be^{-t} \left(-\sin(\sqrt{3}t) + \sqrt{3} \cos(\sqrt{3}t) \right)$$

$$A = 1, -A + \sqrt{3}B = 0 \Rightarrow A = 1, B = \frac{\sqrt{3}}{3}$$

$$y(t) = e^{-t} \cos(\sqrt{3}t) + \frac{\sqrt{3}}{3} e^{-t} \sin(\sqrt{3}t)$$

We now use Maple to plot all three functions for $0 \leq t \leq 2\pi$.

> restart:

$$m := 1$$

$$Om0 := 5.$$

> Y1 := t -> (4/3)*exp(-t) - (1/3)*exp(-4*t);

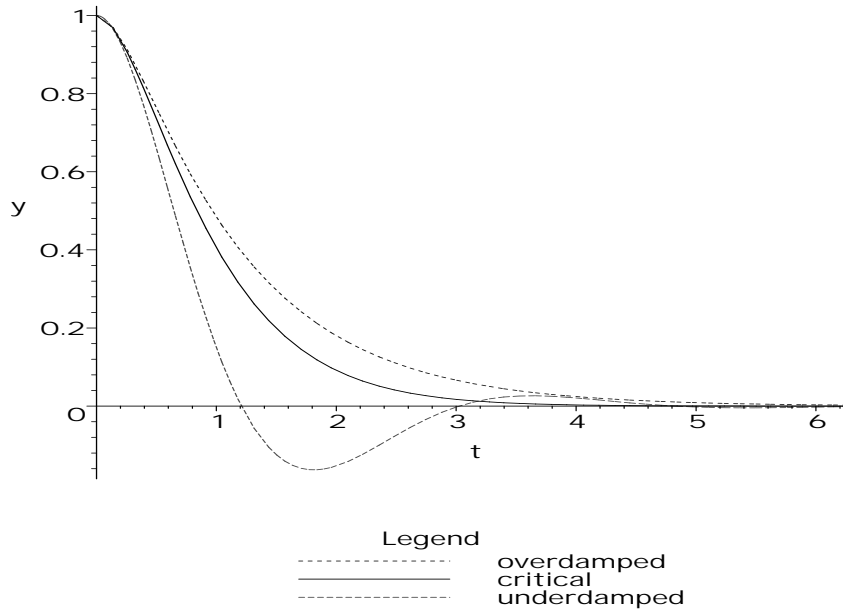
$$Y1 := t \rightarrow \frac{4}{3} e^{(-t)} - \frac{1}{3} e^{(-4t)}$$

> Y2 := t -> exp(-2*t)*(1 + 2*t); Y3 := t -> exp(-t)*(cos(sqrt(3)*t)
+
(1/sqrt(3))*sin(sqrt(3)*t));

$$Y2 := t \rightarrow e^{(-2t)} (1 + 2t)$$

$$Y3 := t \rightarrow e^{(-t)} \left(\cos(\sqrt{3}t) + \frac{\sin(\sqrt{3}t)}{\sqrt{3}} \right)$$

> plot([Y1(t),Y2(t),Y3(t)], t =
> 0..2*Pi,color=[blue,black,red],labels=[t,y],legend=[overdamped,critica
> l,underdamped],linestyle=[DASHDOT,SOLID,DASH]);



4 Problem 4.6.39

Swinging Door: The motion of a swinging door with an adjustment screw that controls the amount of friction on the hinges is governed by the initial value problem

$$I\theta'' + b\theta' + k\theta = 0 ; \theta(0) = \theta_0 , \theta'(0) = u_0 ,$$

where θ is the angle that the door is open, I is the moment of inertia of the door about its hinges, $b > 0$ is a damping constant that varies with the amount of friction on the door, $k > 0$ is the spring constant associated with the swinging door, θ_0 is the initial angle that the door is opened, and u_0 is the initial angular velocity imparted to the door. If I and k are fixed, determine for which values of b the door will not continually swing back and forth when closing.

Solution: Substituting $y(t) = e^{rt}$ as usual we arrive at the characteristic equation for r :

$$Ir^2 + br + k = 0 \Rightarrow r_{\pm} = \frac{-b \pm \sqrt{b^2 - 4Ik}}{2I}$$

It is clear that to avoid oscillations, we must avoid complex roots. The limiting value of b for which oscillations will not happen is the critical value (boundary between overdamped and underdamped cases), i.e. when the discriminant of the quadratic vanishes:

$$b^2 - 4Ik = 0 \Rightarrow b_{critical} = 2\sqrt{Ik}$$

when $r = -b/2I$ and all values of $b \geq b_{critical}$ give non-oscillatory motion.